

变厚度薄板弯曲问题的任意网格差分解法

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摘 要

本文提出一种求解变厚度薄板弯曲问题的任意网格差分格式, 可适应不同边界, 各种荷载和复杂形状板. 计算实例表明, 该方法具有格式简单、通用性强、计算精度高、计算量少等特点.

关键词 有限差分法 变厚度薄板弯曲问题

一、引 言

有限差分法由于格式简单, 计算效率高, 一直是求解薄板弯曲问题最重要的数值方法. 但经典差分法由于受正规网格剖分的限制, 不能方便地处理非规则边界条件, 近年来, 国内外出现了一些求解薄板弯曲问题的非规则网格差分法^[1~8], 目前这些研究大多局限于等厚度薄板.

本文提出一种求解变厚度薄板弯曲问题的任意网格差分格式, 可考虑各种边界条件, 荷载情况和复杂边界. 为验证本文方法的精度和可靠性, 本文给出了若干计算实例, 数值结果表明本文方法不失为一种简明有效的方法.

二、基本方程

变厚度薄板弯曲问题的平衡方程为

$$\begin{aligned} D(x,y)\nabla^4 w + 2 \frac{\partial D(x,y)}{\partial x} \frac{\partial}{\partial x} \nabla^2 w + 2 \frac{\partial D(x,y)}{\partial y} \frac{\partial}{\partial y} \nabla^2 w \\ + \left[\frac{\partial^2 D(x,y)}{\partial x^2} + \mu \frac{\partial^2 D(x,y)}{\partial y^2} \right] \frac{\partial^2 w}{\partial x^2} \\ + \left[\frac{\partial^2 D(x,y)}{\partial y^2} + \mu \frac{\partial^2 D(x,y)}{\partial x^2} \right] \frac{\partial^2 w}{\partial y^2} \\ + 2(1-\mu) \frac{\partial^2 D(x,y)}{\partial x \partial y} \frac{\partial^2 w}{\partial x \partial y} = q(x,y) \end{aligned} \quad (2.1)$$

其中: $D(x, y) = \frac{Eh^3(x, y)}{12(1-\mu^2)}$, $h(x, y)$ 为板的

厚度, 基本边界如图1所示, 边界条件为如下几种形式:

(1) 简支边 Γ_s

$$w = \bar{w} \quad (2.2)$$

$$-D \left[\frac{\partial^2 w}{\partial x^2} (\cos^2 \theta + \mu \sin^2 \theta) + \frac{\partial^2 w}{\partial y^2} (\sin^2 \theta + \mu \cos^2 \theta) \right. \\ \left. + 2(1-\mu) \frac{\partial^2 w}{\partial x \partial y} \sin \theta \cos \theta \right] = \frac{\partial \bar{w}}{\partial n} \quad (2.3)$$

(2) 固支边 Γ_c

$$w = \bar{w} \quad (2.4)$$

$$\frac{\partial w}{\partial x} \cos \theta + \frac{\partial w}{\partial y} \sin \theta = \frac{\partial \bar{w}}{\partial n} \quad (2.5)$$

(3) 自由边 Γ_f

$$-D \left[\frac{\partial^2 w}{\partial x^2} (\cos^2 \theta + \mu \sin^2 \theta) + \frac{\partial^2 w}{\partial y^2} (\sin^2 \theta + \mu \cos^2 \theta) \right. \\ \left. + 2(1-\mu) \frac{\partial^2 w}{\partial x \partial y} \sin \theta \cos \theta \right] = \bar{M}_n \quad (2.6)$$

$$D \left\{ (1-\mu) \frac{\partial}{\partial s} \left[\left(\frac{\partial^2 w}{\partial x^2} - \frac{\partial^2 w}{\partial y^2} \right) \sin \theta \cos \theta - \frac{\partial^2 w}{\partial x \partial y} (\cos^2 \theta - \sin^2 \theta) \right] \right. \\ \left. - \left(\frac{\partial^3 w}{\partial x^3} + \frac{\partial^3 w}{\partial x \partial y^2} \right) \cos \theta - \left(\frac{\partial^3 w}{\partial y^3} + \frac{\partial^3 w}{\partial x^2 \partial y} \right) \sin \theta \right\} = \bar{V}_n \quad (2.7)$$

(4) 角点 Γ_k

$$\left(\frac{\partial^2 w}{\partial x^2} - \frac{\partial^2 w}{\partial y^2} \right) \cos(\theta_+ + \theta_-) + 2 \frac{\partial^2 w}{\partial x \partial y} \sin(\theta_+ + \theta_-) = 0 \quad (2.8)$$

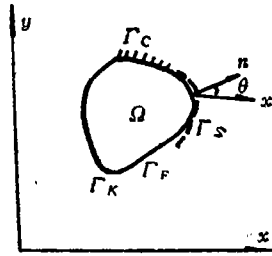


图 1

三、差分逼近

对可微函数 $f(x, y)$, 其 Taylor 展开式为:

$$f = f_0 + \sum_{k=1}^n \frac{1}{k!} \left(h_x \frac{\partial}{\partial x} + h_y \frac{\partial}{\partial y} \right)^k f_0 + O(h^{n+1}) \quad (3.1)$$

为得到四阶导数项的差分系数, 在方程(3.1)中取 $n=4$, 得

$$f_i = f_0 + \sum_{k=1}^4 \frac{1}{k!} \left(h_x \frac{\partial}{\partial x} + h_y \frac{\partial}{\partial y} \right)^k f_0 \quad (3.2)$$

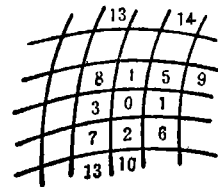


图 2

其中 (x_i, y_i) 为 $(i=1, 14)$ 为 (x_0, y_0) 的邻点, 如图2所示, $h_{xi}=x_i-x_0$, $h_{yi}=y_i-y_0$. 对方程(3.2)加权求和, 可得

$$\sum_{i=1}^{14} \alpha_i (f_i - f_0) = \sum_{i=1}^{14} \alpha_i \sum_{k=1}^4 \frac{1}{k!} \left(h_{xi} \frac{\partial}{\partial x} + h_{yi} \frac{\partial}{\partial y} \right)^k f_0 \quad (3.3)$$

为得到平衡方程(2.1)的差分格式, 在方程(3.3)中, 令:

$$\left. \begin{aligned} \sum_{i=1}^{14} \alpha_i h_{xi} &= 0, \quad \sum_{i=1}^{14} \alpha_i h_{yi} = 0 \\ \sum_{i=1}^{14} \alpha_i h_{xi}^2 &= 2 \left(\frac{\partial^2 D}{\partial x^2} + \mu \frac{\partial^2 D}{\partial y^2} \right)_0 \\ \sum_{i=1}^{14} \alpha_i h_{yi}^2 &= 2 \left(\frac{\partial^2 D}{\partial y^2} + \mu \frac{\partial^2 D}{\partial x^2} \right)_0 \\ \sum_{i=1}^{14} \alpha_i h_{xi} h_{yi} &= 2(1-\mu) \left(\frac{\partial^2 D}{\partial x \partial y} \right)_0 \\ \sum_{i=1}^{14} \alpha_i h_{xi}^3 &= 12 \left(\frac{\partial D}{\partial x} \right)_0, \quad \sum_{i=1}^{14} \alpha_i h_{yi}^3 = 12 \left(\frac{\partial D}{\partial y} \right)_0 \\ \sum_{i=1}^{14} \alpha_i h_{xi}^2 h_{yi} &= 4 \left(\frac{\partial D}{\partial y} \right)_0, \quad \sum_{i=1}^{14} \alpha_i h_{xi} h_{yi}^2 = 4 \left(\frac{\partial D}{\partial x} \right)_0 \\ \sum_{i=1}^{14} \alpha_i h_{yi}^4 &= 24(D)_0, \quad \sum_{i=1}^{14} \alpha_i h_{xi}^4 = 24(D)_0 \\ \sum_{i=1}^{14} \alpha_i h_{xi}^2 h_{yi}^2 &= 8(D)_0, \quad \sum_{i=1}^{14} \alpha_i h_{xi}^3 h_{yi} = 0 \\ \sum_{i=1}^{14} \alpha_i h_{xi} h_{yi}^3 &= 0 \end{aligned} \right\} \quad (3.4)$$

求解线性方程组(3.4)得加权系数 α_i , 由方程(3.3)即可得平衡方程(2.1)的差分格式为:

$$\sum_{i=1}^{14} \alpha_i (w_i - w_0) = q_0 \quad (3.5)$$

类似可得到各类边界条件的差分格式. 由于采用曲四边形剖分, 可方便地处理非规则域边界条件和实现网格的局部加密.

四、计算实例

利用上述公式和数值方法编制了变厚度薄板弯曲问题的有限差分通用程序MFP, 下面

为该程序计算的几个计算实例。

4.1 四边简支方板

如图3所示的四边简支方板受分布荷载 $q=q_0y/a$

作用, 板厚为 $h(x,y) = \left[1 + \lambda \left(\frac{2y}{a} - 1\right)\right] h_0$, h_0

为 $y=a/2$ 处厚度, 取 $\lambda=0.2$, $\mu=0.25$ 划分为 8×8 网格, 12×12 网格计算挠度和弯曲矩计算值如表1和图4所示:

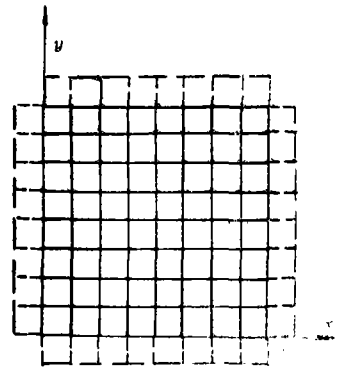


图 3

表 1

项目 数值 y/a	$w\left(\frac{a}{2}, y\right)$			$M_y\left(\frac{a}{2}, y\right)$			$M\left(\frac{a}{2}, y\right)$		
	8×8网格	12×12网格	级数 ^[12]	8×8网格	12×12网格	级数 ^[13]	8×8网格	12×12网格	级数 ^[13]
0.25	0.001458	0.001443	0.001439	0.01243	0.01240	0.01255	0.01377	0.01375	0.01325
0.5	0.001984	0.001970	0.002000	0.02233	0.02231	0.02304	0.02250	0.02250	0.02337
0.75	0.001407	0.001398	0.001439	0.02111	0.02109	0.02238	0.02259	0.02265	0.02419

表中挠度和弯矩值分别为 q_0a^4/D_0 和 q_0a^2

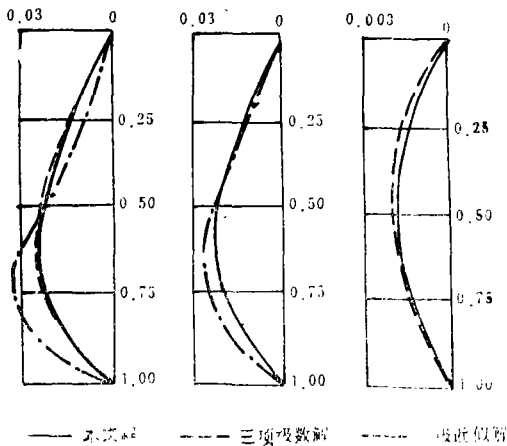


图 4

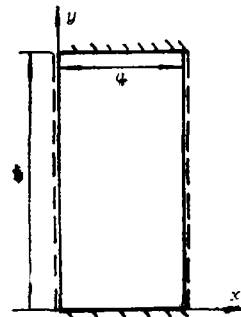


图 5

4.2 对边简支对边固支矩形板

如图5所示的对边简支对边固支矩形板, 受分布荷载 $q=q_0 \exp[2y/b] \sin(\pi x/a)$ 作用, 弯曲刚度为 $D=D_0 \exp[2y/b]$, $a=0.5m$, $b=1m$. 数值结果如表 2、3 所示。

4.3 不同边界条件的圆板

半径为 a 的圆薄板, 板厚变化规律为: $h=h_0 \exp[-\beta(r/2)^2/6]$, 受均布荷载 q_0 作用, 计算了固支边和简支边两种情形, 剖分如图6所示, 表 4 和表 5 分别给出了固支和简支边界条件下 β 取不同值时的中心挠度值。

表 2

$$w(a/2, y) [(q_0 b^4 / D_0) \times 10^{-4}]$$

数值 y/a	8×8网格	12×12网格	16×16网格	精确解[9]
0.25	3.52685	3.31049	3.13741	3.11158
0.5	4.40141	4.25459	4.14483	4.14551
0.75	2.65537	2.51021	2.39408	2.38703

表 3

$$M_y \left(\frac{a}{2}, y \right) q_0 b^2$$

数值 y/a	8×8网格	12×12网格	16×16网格	精确解[9]
0	-0.023750	-0.027820	-0.031076	-0.032480
0.25	0.014580	0.014367	0.014298	0.014260
0.5	0.022928	0.022936	0.022945	0.023175
0.75	0.021482	0.021031	0.020671	0.020687
1.0	-0.110097	-0.121836	-0.131227	-0.134087

表 4

固支圆板中心点挠度值 (乘子: qa^4/D_0)

β	4	2	-2	-4
级数解 ^[1]	0.04005	0.02525	0.00960	0.00597
差分解	0.04087	0.02569	0.00973	0.00607

表 5

简支圆板中心点挠度值 (乘子: $D_0^{-1}qa^4$)

β	4	2	-2	-4
级数解 ^[10]	0.11165	0.08460	0.04685	0.03302
差分解	0.09860	0.08396	0.04505	0.03218

表中: $D_0 = Eh^3/12(1-\mu^2)$

五、结 论

从以上算例可以看出, 本文提出的任意网格差分法具有计算简单, 计算量少, 计算精度高, 通用性强等特点, 有限元法求解薄板弯曲问题时, 其节点自由度除去挠度外, 还有转角, 因而在相同的网格下扩大了计算规模。因此, 本文方法通用性与有限元法相当, 但计算量远少于有限元法。

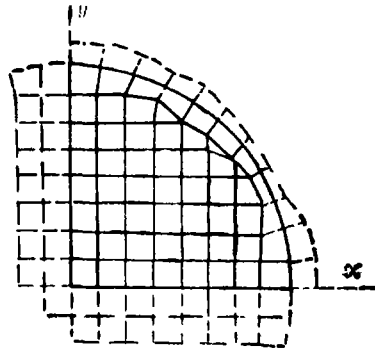


图 6

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A Finite Difference Method at Arbitrary Meshes for the Bending of Plates with Variable Thickness

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Abstract

A finite difference method at arbitrary meshes for the bending of plates with variable thickness is presented in this paper. The method is completely general with respect to various boundary conditions, load cases and shapes of plates. This difference scheme is simple and the numerical results agree well with those obtained by other methods.

Key words finite difference method, bending of plates with variable thickness